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On the determination of the shock and steady state parameters of gelatine from cylinder impact experiments

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A B S T R A C T

For a soft body projectile striking a target or the shock loading of a soft body material, the determination of the interface shock pressure, shock speed and applied steady state pressures is important but has been hindered by technical challenges even with the use of sophisticated embedded pressure sensors in the target surface. Difficulties interpreting the results render the accuracies sometimes questionable or impossible to reproduce. Here we propose a simple impact experiment using a force sensor and an analysis procedure to derive the interface pressure from the force/time history. The results are compared to those obtained from shock Hugoniot and penetration equations. We came upon the presence of a dynamic pressure that is significantly higher than the expected stagnation pressure. This method could be used to determine and characterise the shock and steady state pressures of a wider range of materials under impact and shock loading conditions.

1. Introduction

Low strength materials such as water, gelatine, rubber, wax or even emulsions are used for a broad range of applications involving impact and shock loading conditions. Apart from being used as shock absorbers or in energy dissipating systems, these materials, such as gelatine and rubber, are used as surrogates for human body tissues, organs, biological liquids, animals and birds to examine the effects that may occur due to impact or shock. Examples range from the use of extra-corporeal shock wave lithotripsy [1] in the non-invasive disintegration of urinary tract stones or ultrasounds for the denaturing of deep seated cancerous cells [2] to the studying of trauma [3] caused to the human body due to impact or shock loading or the damage to aircraft structures due to bird impact [4–12]. In all these examples, the shock and steady state pressures are important loading parameters that are needed to understand the response of the materials and these parameters are normally measured at the impactor/target interface. In an impact problem, especially in the case involving a soft body material, the simultaneous deformation of the projectile and target makes uncoupling the response of each material very difficult so to decouple and understand the responses, studying cylinders made out of a particular material striking rigid targets provides researchers with a very useful means to characterise a material under shock loading condition whether it is used as an impactor as in the case of a bird strike problem or a target as in the case of a human surrogate struck by a projectile. Studies [13–19] on the

deformation of solids by liquid impact at supersonic speeds examined flat-ended cylinders striking a rigid target and described the interface pressure as the water hammer pressure, $P = \rho c_0 u_0$ where ρ is the density, c_0 is the wave speed and u_0 is the impact velocity. Further studies [24–28] on the issue have shown that the water hammer equation pressure works well only for low velocity impact but for higher velocities, c_0 must be replaced by the shock velocity, U_s, to get what is called the shock or Hugoniot pressure, $P_h = \rho U_s u_0$. These studies have all shown that when a projectile strikes a target (Fig. 1) a shock is generated at the center of the projectile and propagates towards the outside surface and on reflection, forms release waves that propagate towards the center at a lower pressure which causes the material to flow. The pressure at the interface begins to decrease and after several reflections the projectile flow will approach a steady state condition where the pressure becomes the stagnation pressure, $P_{stagnation} = \frac{1}{2}\rho u_0^2$. Many studies [4,7–9,13–24] have validated this theory of the shock and steady state regimes govern, respectively, by the shock Hugoniot and the steady state pressure and from the literature cited there is general agreement on this. Many researchers [4–12] in measuring these pressures use pressure transducers embedded in the surface of the target where the projectile first strikes. However, in all the work cited for soft body impact, although good shock and steady state pressure results are obtained, there are many difficulties, such as the limitation of the pressure gauges that rendered the accuracies of the data sometimes being questionable or difficult to reproduce.
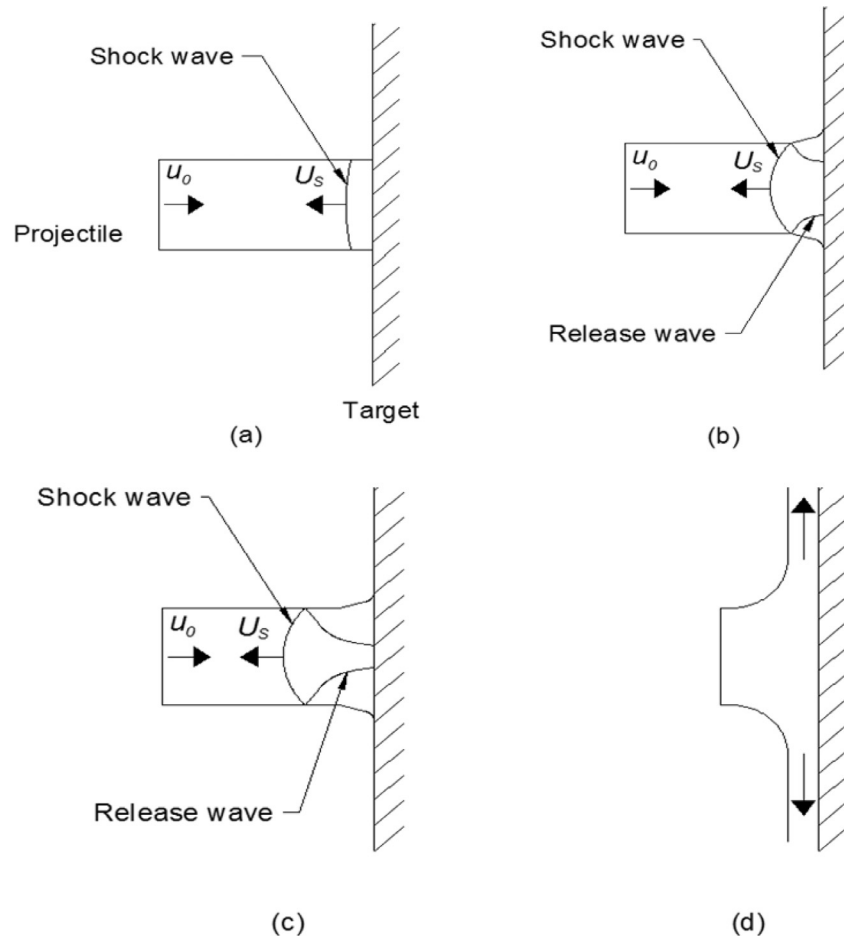
Fig. 1. Illustration of the four stages of a soft body projectile striking a rigid target.

Studies [4–6,10,11] spanning the last three decades have highlighted these limitations and this led us to re-look at the issues of the impact problem and examine whether reliable shock and steady state pressures could be obtained from a simple more repeatable experiment that could be used to examine a broad range of soft body materials.

2. Experiments

Here we propose using a force ring transducer, instead of a pressure gauge, sandwiched between a rigid target disc and a clamped support plate to determine the interface pressure using the force-time history. An air gun impact facility (Fig. 2(a)) was used to conduct the experiments. The air gun itself consisted of a 1.8-m long, 40-mm diameter launch tube that is coupled to a compressed air reservoir. The pressure of the air in the reservoir determines the exit velocity of the projectile. A phototron high speed camera recording at 20,000 frames/s, was used to record the projectile release from the launch package, its flight to and interaction with the target. The data acquisition system integrated with the gun firing mechanism, was used to trigger the camera and acquire the force history. To launch projectiles with the air gun, a sabot (Fig. 2(b), Appendix A.2 - Sabot development and Appendix B - Fig. B. 1) was required to hold the penetrator in place during its travel in the launch tube and then stripped away before striking the target. The basic projectile (Fig. 2(b)) was a 28-mm cylindrical 10% gelatine rod with a hemispherical tip and a nominal length of 102 mm and was prepared using a standard 10% gelatine recipe [1,9] (Appendix A.1 - Gelatine

preparation). The target was a 120-mm diameter, 19-mm thick steel disc with a solid 28-mm diameter cylindrical support at the center that was attached to the center of a 330-mm, 35-mm thick square steel plate sandwiching a force ring sensor (Fig. 2(c)). The force transducer used was a PCB Piezotronics Quartz Force Ring Sensor Model 207C (Fig. 2(d)) with a force measurement of up to 445 kN and a sensitivity of ± 1.5%.

## 3. Results - experimental data reduction and discussion

Of the many tests conducted, four at different impact velocities, 74, 105, 115 and 119 m/s, were chosen to conduct the analysis. Fig. 3 shows a sequence of the projectile/target interaction in time for the 119 m/s impact velocity case, starting with the projectile striking the target until it was completely eroded. A closer look at the sequence of pictures reveals that as the material goes from the initial shock phase and into deformation due to the large compressive forces, the front of the projectile mushrooms and then there appears to be considerable shearing and fissuring of the material into fragments and subsequently entering into the radial flow which remains parallel to the target. This tearing or shearing of the material into fragments are also very evident from the pieces of the gelatine (Appendix B - Fig. B. 2) gathered after the test. This appears to be different than the flow of water or a liquid striking the target. The solid line of the force histories shown in Fig. 4 are the raw force data as acquired from the tests and significant oscillations were observed in the results. A Fourier transform [21] on the
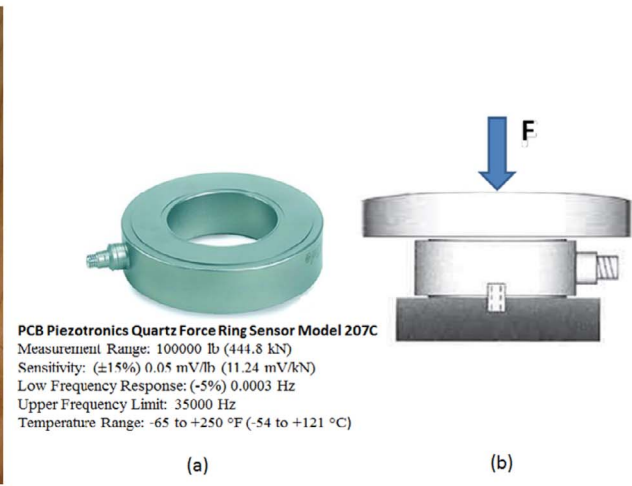
(a) Test setup.
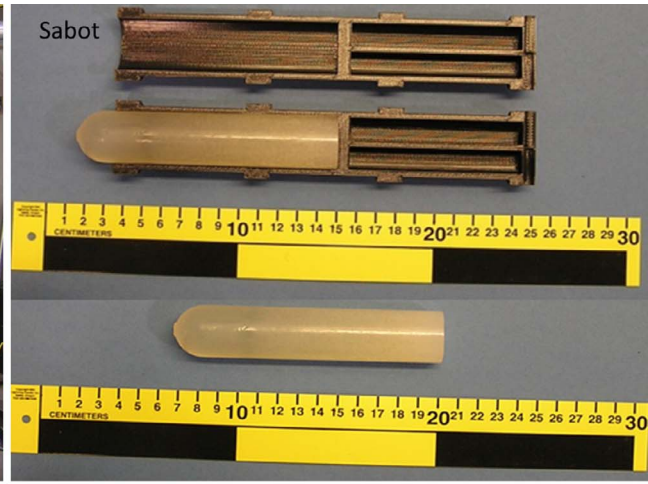
(b) Gelatin projectile and sabot.
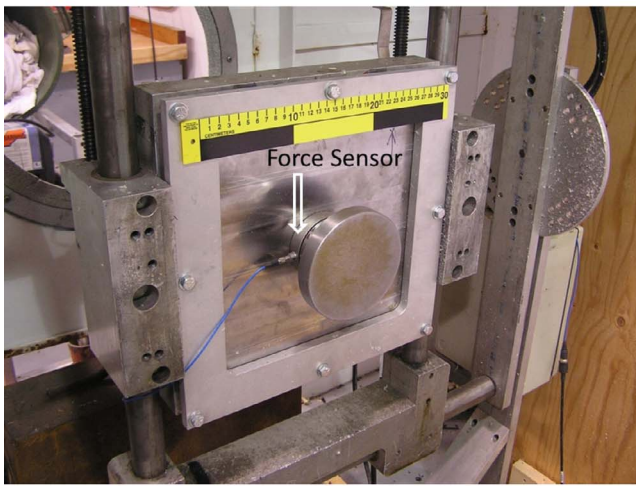
(c) Target with force sensor installed.

(d) Force sensor and configuration.

Fig. 2. Air gun, gelatine projectile, target and force transducer setup.

force time data (Appendix B - Fig. B. 3) revealed the presence of two frequencies one at 2000 Hz and the other at 15,000 Hz and whose amplitudes changed significantly with the impact velocity. A manual roving hammer test was conducted, striking different parts of the target and its supporting structure and the two frequencies were generated only when either the target disc or supporting plate was struck. To confirm the source of these frequencies, the natural frequencies of the two plates were calculated using the Bessel function solution [22–24] to the wave equation for the vibrations of plates, $f = \frac{1}{2\pi} \frac{\lambda^2}{a} \sqrt{\frac{D}{\rho h}}$ where $D = \frac{Eh^3}{12(1-\nu^2)}$ is the flexural rigidity. ρ is the density, E is the Young's modulus, h is the plate thickness, and ν is the Poisson's ratio. a is either the diameter of the disc or length of plate. λ^2, the non dimensional Bessel coefficient [26–28] for the first mode, equals 36 for a clamped square plate and 6.25 for an annular disc clamped at the center. The target disc was treated as an annulus [29], free on the outside and clamped on the inside over a 28-mm diameter area where it is attached to the square plate sandwiching the Force ring sensor. The frequencies calculated were very close to the frequencies found in the Fourier

transform of the force time data confirming that these two superimposed frequencies were from the free vibrations of the target plates (Appendix B - Fig. B. 3). The Savitsky–Golay [30,31] filter that tends to preserve key data features such as peak height, width and zero phase shift so that the signals are not shifted, was used to remove the 15,000 Hz frequency whereas a Butterworth band-stop filter was used to remove the lower 2000 Hz frequency. The dashed curves in Fig. 4 show the filtered data. The force caused by the shock pressure is taken to be the maximum force arrived at just after the steep rise which is essentially the highest peak in the raw data history. The remainder of the force-time curve would indicate the steady state loading of the target until the projectile has completely eroded or rebounded. In the 74 m/s impact velocity case, two peaks in the force-time curve could be observed. The video showed that the projectile broke into two pieces before impact and essentially there were two impacts one after the other occurring at the target. The impact velocity of the second part may have been be a little lower than 74 m/s given that it seemed to have struck the back end of the first part of the projectile before coming into contact with the target. The results, therefore, appears to be
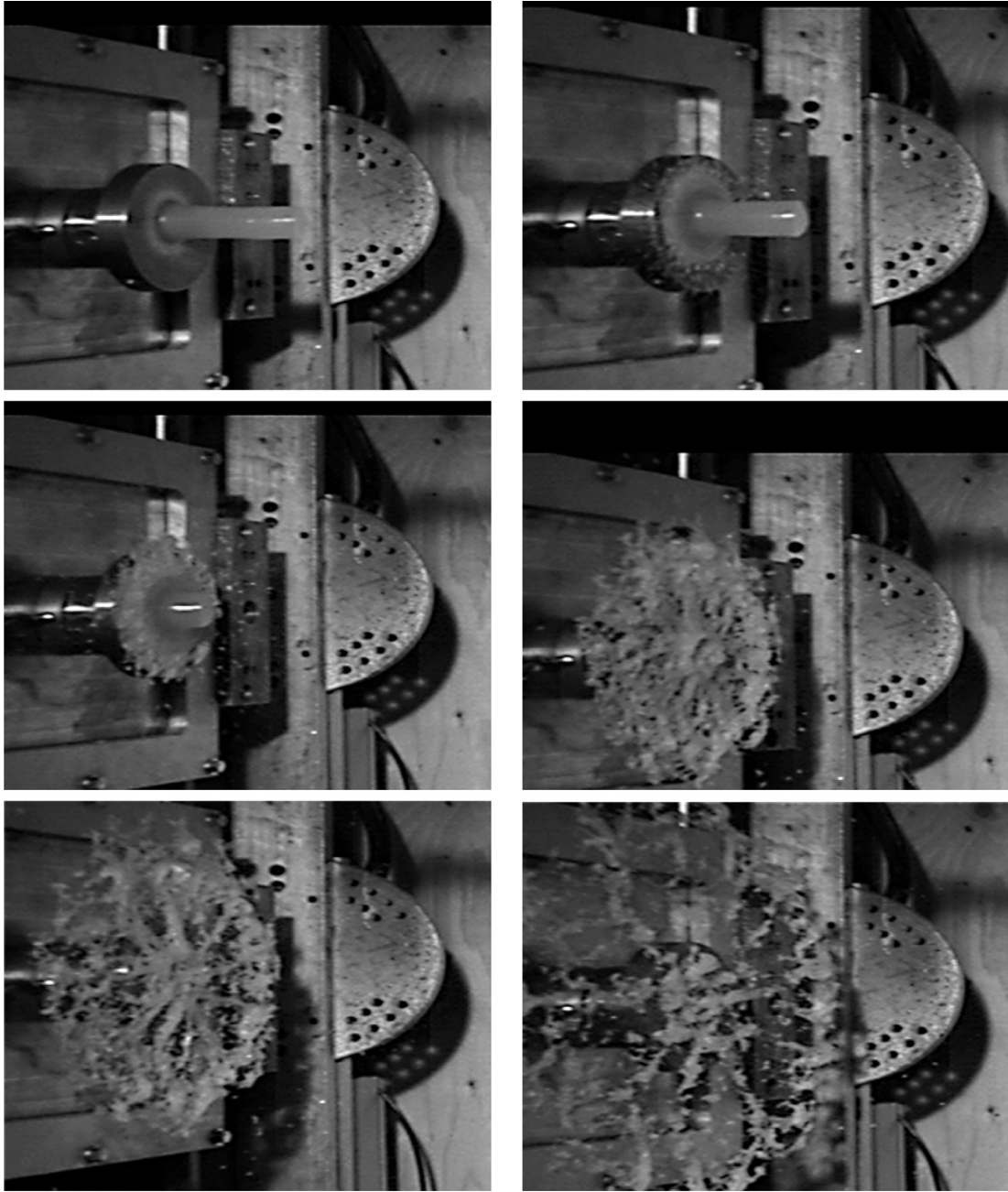
Fig. 3. Time sequence snapshots of the projectile/target interaction for the 119 m/s impact velocity test case.

consistent with those of the other tests. In an impact problem, when a pressure is applied, based on the difference in the impedances of the two materials, part of the pressure is transmitted and the part is reflected [1,32,33]. It is known [1] that the fraction that is transmitted is given by the transmission coefficient, T_t, as

$$T_t = \frac{2Z_2}{Z_1 + Z_2} \tag{1}$$

whereas, for the reflected fraction, the reflection coefficient T_r, is given by

$$T_r = \frac{Z_2 - Z_1}{Z_1 + Z_2} \tag{2}$$

where $Z_1 = \rho_1 U_{s1}$, $Z_2 = \rho_2 U_{s2}$ are the shock impedances for medium 1 and 2, respectively, and U_{s1} and U_{s2} are the corresponding shock speeds which are obtained from the $U_s - u_p$ shock Hugoniot (Appendix A.3 - Shock pressure solution and Appendix B - Fig. B. 4)

$$U_s = c_0 + s u_p \tag{3}$$

where c_0 is the acoustic velocity and s is the slope. $T_t + T_r = 1$ and because of this condition, medium 2 would be assigned to the steel target
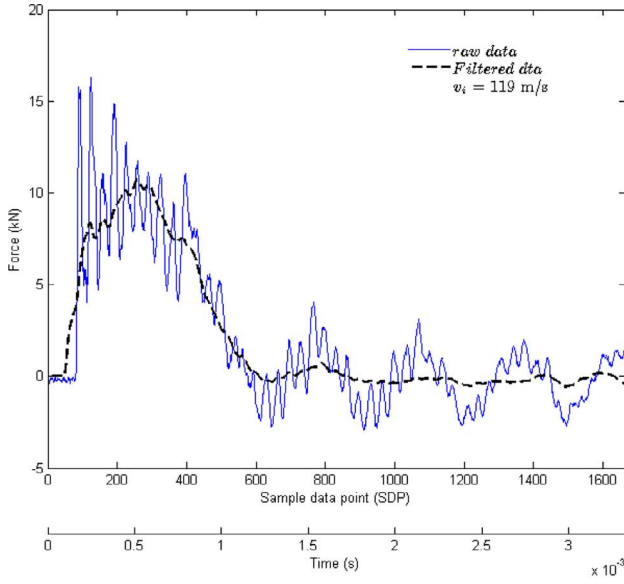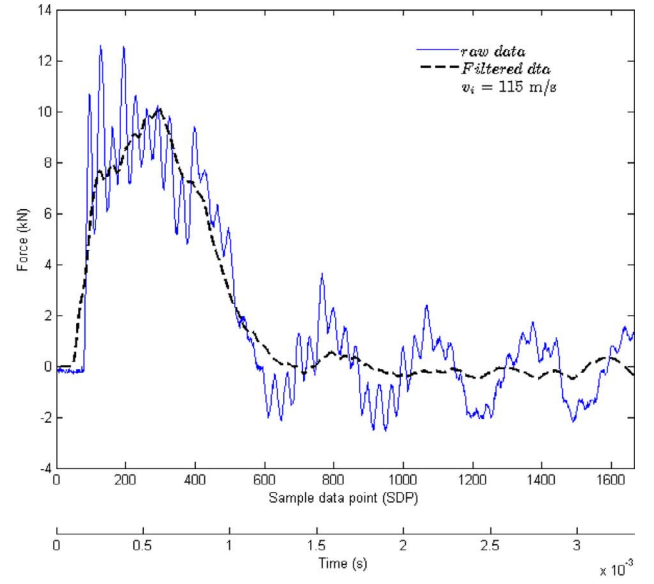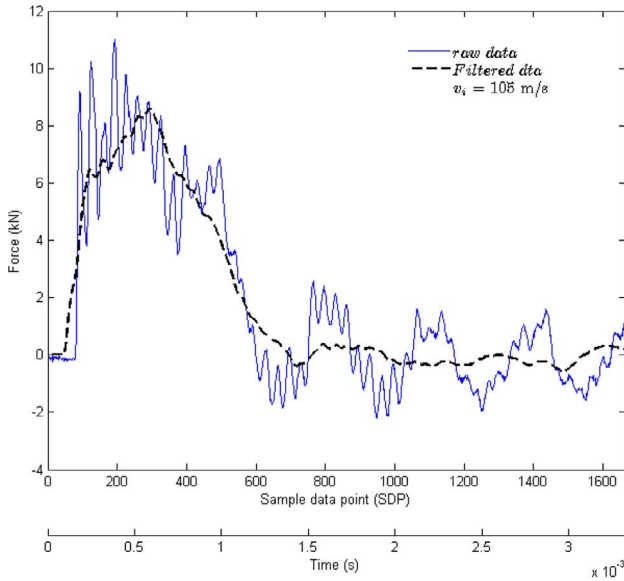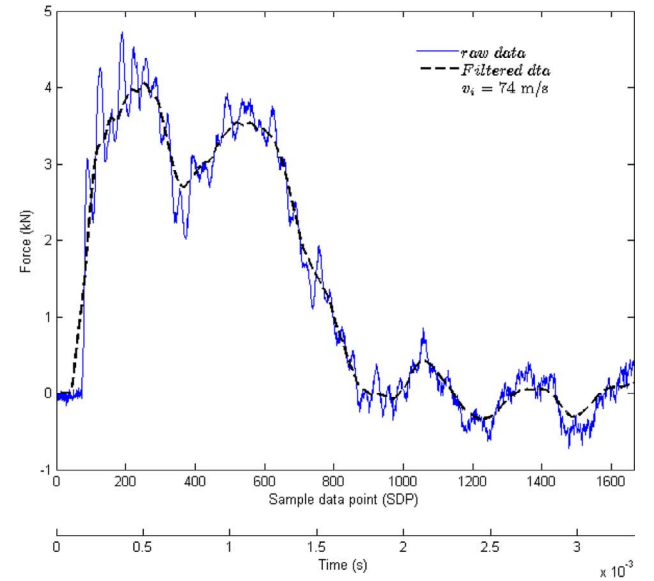
(a) Force history $v_i = 119$ m/s.

(b) Force history $v_i = 115$ m/s.

(c) Force history $v_i = 105$ m/s.

(d) Force history $v_i = 74$ m/s.

Fig. 4. Raw and filtered force history as a function of sampling data point and time.

Table 1
Properties/parameters required to determine shock and particle velocities in gelatine [3] and steel [32].

Property/parameter	Gelatine	Steel
Density, ρ (kg/m^3)	1030	7800
Acoustic velocity, c_0 (m/s)	1445	5800
Shock parameter, s	1.9	1.434

which is the higher impedance material of the two materials that are in contact and medium 1 would be assigned to the gelatine. Thus, the amplitude of the force measured would be caused by the transmission fraction, T_t, of the original pressure applied at the projectile/target

interface. The shock properties [3,32] required to calculate the shock speeds in the gelatine and steel are given in Table 1.

Table 2 shows a summary of the important results for the four tests conducted. For each impact velocity, u_{0p} or v_i, the corresponding interface particle velocity, u_p was calculated (Appendix A.3 - Shock pressure solution, Eq. (A.6)). This was then used to calculate the shock speeds, U_{steel} and $U_{gelatine}$ using Eq. (3). The shock speeds were then used to calculate T_r and T_t using Eqs. (1) and (2). Knowing the force, F_h (Table 2), the transmission coefficient is used to determine the interface shock pressure, P_{he} (Table 2). For example, for the case where $v_i = 119$ m/s, $T_t = 0.073$. Thus, 7.3% of the of the interface shock pressure was transmitted. Scaling this to 100% and dividing by the cross sectional area of the projectile, 616 mm^2, an interface pressure, P_{he}, of 191 MPa

Table 2

A summary of results giving the impact velocity, v_i, the interface particle velocity, u_p, shock speed in steel target and gelatine projectile, U_{steel} and $U_{gelatine}$, the transmission and reflective coefficients, T_t and T_r, the filtered experimental shock force, F_h, the experimentally derived shock pressure P_{he} and the theoretical shock pressure, P_{ht}. t_{exp} and t_{pr} are, respectively, the experimental and predicted projectile/target interaction time.

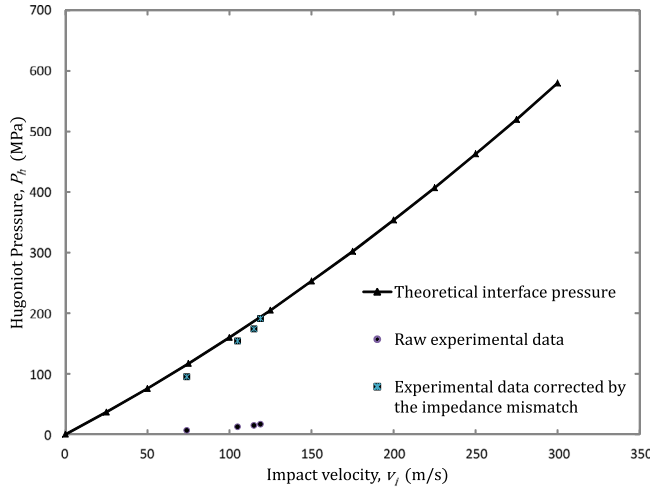| v_i (m/s)	u_p (m/s)	U_{steel} (m/s)	$U_{gelatine}$ (m/s)	t_{exp} (ms)	t_{pr} (ms)	T_r	T_t	F_h (kN)	P_{he} (MPa)	P_{ht} (MPa)
119	3.8	5806	1663	0.87	0.88	0.927	0.073	8.59	191	196
115	3.8	5806	1655	0.90	0.89	0.927	0.073	7.73	174	189
105	3.8	5805	1637	1.00	0.98	0.923	0.072	6.80	154	171
74	2.7	5804	1581	1.40	1.38	0.931	0.069	4.05	95	116

Fig. 5. Comparison of the calculated experimental shock pressure with the theoretical value as a function of impact velocity.

was obtained. Fig. 5 shows a comparison of the theoretical, $P_{ht} = \rho U_s v_i$ (Appendix A.3 - Shock pressure solution, Eq. (A.7)) and the derived experimental interface shock pressures P_{he}, as a function of impact velocity and good agreement is obtained.

For the duration of the projectile/target interaction (Table 2) there is very good agreement between the measured time, t_{exp} and the predicted value $t_{pr} = \frac{l}{v_i}$ obtained from the Alekseevskii–Tate equations [25,26,34,37,38] (Appendix A.4- Steady state solution, Eq. (A.13)). This is an indication that the erosion rate and the application of the pressure on the target are consistent with the measured force. With respect to the steady state interface pressure, in accordance with the penetration equations (Appendix A.4 - Steady state solution, Eqs. (A.9) and (A.14)), it was established that if Y_p, the projectile dynamic strength, is assumed to be negligible then the steady state pressure at the interface would be $P_{stagnation} = \frac{1}{2}\rho v_i^2$. However, interpretation of the experimental results does not appear to indicate this. Consider the experimental results shown in Fig. 6, $P_{stagnation}$ is plotted on the same axes as the experimentally derived pressure-time curves.

Examination of the two curves reveals that the $P_{stagnation}$ is much lower than the experimentally derived pressure by an amount, ΔP. This indicates the presence of a dynamic pressure that the projectile material is applying to the target and is of the order of the shock pressure. This observation could be explained in accordance with the penetration equations (Appendix A.4 - Steady state solution, Eq. (A.14)). If Y_p is taken into consideration [35], then setting $Y_p = \Delta P$, Y_p could be termed as a dynamic pressure applied by the gelatine and it is impact velocity

dependent. We know that this could not be due alone to the strength of the projectile as is often used in the case of other solid penetrators materials because studies [40,41] have shown that the fracture strength of gelatine at reasonably high strain rates is between 1 and 2 MPa so there must other phenomena occurring. A close observation of the projectile/target interface provides one explanation. It appears that the gelatine material in the mushrooming region of the projectile is undergoing severe compression enough to cause a significant increase in the bulk modulus which in turn manifests itself in a localised increase in the bulk density enough to cause a significant rise in the pressure applied to the target. This continues for about half way through the length of the projectile interaction with the target after which the interface pressure dropped linearly to zero as the projectile erodes away. This argument of the compression in the impact region resulting in presence of this dynamic high pressure could be substantiated by similar observations made by Field et al. [16,17] and other studies [13–24] that examined the impact of liquid and water-gelatine mixtures on solid structures only in this case we do have a quantitative measure of this dynamic high pressure present at the interface. Fig. 7 shows the maximum ΔP as a function of impact velocity and a quadratic fit to the data indicates that the peak dynamic strength pressure at the interface rapidly increases with impact velocity.

4. Conclusion

A simple cylinder impact test was used to examine the impact process of a soft body material projectile made of 10% gelatine, striking a rigid target attached to a quartz force ring sensor instead of a usually used pressure sensor. The shock and steady state pressures were determined from the force-time histories. In an effort to obtain accurate and reproducible results, the force time curves were examined to identify frequencies that were present in the results and may be coming from the natural vibrations of the various parts that make up the target. The Bessel function solutions for the wave equation applied to the free vibration of plates were used to examine the vibrations of the various plates in the target assembly and once identified and determined were removed using the Savitsky–Golay filter. Using the shock Hugoniot relations and the conservations equations, the theoretical shock pressure relation was developed for the shock pressure at the projectile/target interface. The shock pressures derived from the experimental force/time histories were in very good agreement with the theoretical values predicted by the shock Hugoniot. The Alekseevskii–Tate penetration equations were used to determine the steady state pressure at the projectile/target interface and the duration of the interaction between the projectile and target. The equations predicted very well the duration of the interaction. However, the measured value of the steady state pressure was in discordance with the expected stagnation
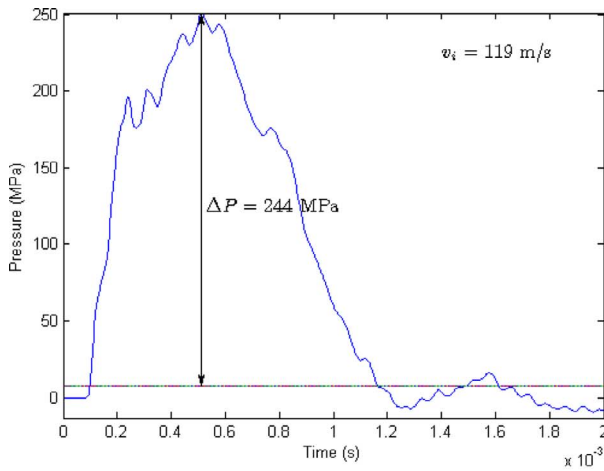
(a) Dynamic interface pressure ΔP, for $v_i = 119$ m/s.
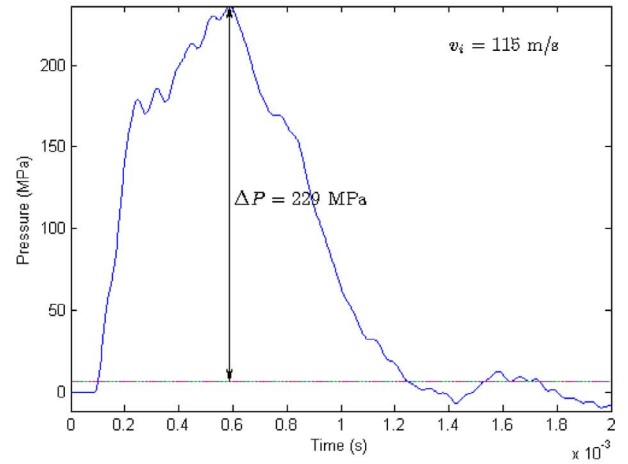
(b) Dynamic interface pressure ΔP, for $v_i = 115$ m/s.
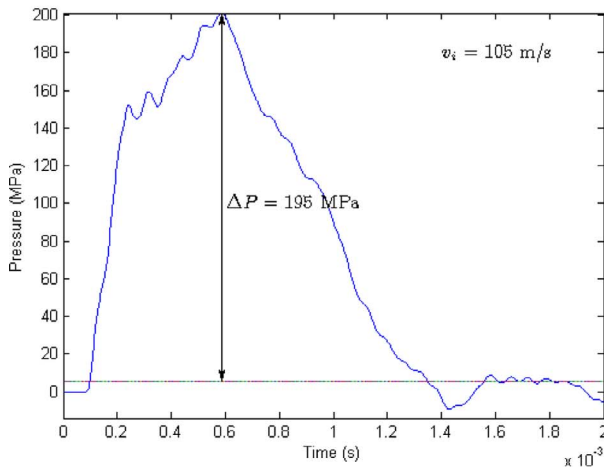
(c) Dynamic interface pressure ΔP, for $v_i = 105$ m/s.
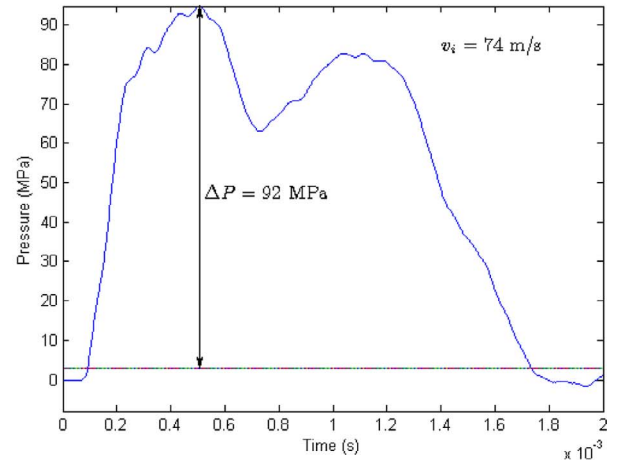
(d) Dynamic interface pressure ΔP, for $v_i = 74$ m/s.

Fig. 6. Dynamic interface pressure, ΔP, as a function of time.
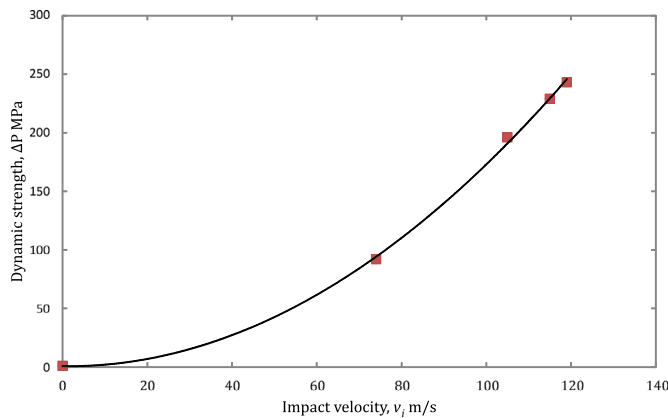
Fig. 7. Maximum interface dynamic pressure, ΔP, as a function of time.

pressure, $P_{stagnation} = \frac{1}{2}\rho v_i^2$. We came upon the presence of a dynamic pressure which when interpreted in accordance with the Alekseevskii–Tate equations, suggests that the material dynamic strength manifested from the compression of the gelatine material caused a change in the bulk modulus or density and, thus, the increased pressure. A quantitative measure of this dynamic strength as a function

of time was obtained and it is dependent on the impact velocity. Given the significant value of this dynamic pressure, it may be necessary to take this into account when these materials are used in shock loading and impact scenarios.

Appendix A. Methods

A1. 10% gelatine preparation

The gelatine projectiles were prepared using a standard recipe [4,10,11]. The ingredients were 1000 g cold water, 100 g ballistic gelatine powder, 25 g sodium carboxy methyl cellulose(CMC), 6 g aluminum acetate basic (AAB), 4 drops of cinnamomum zeylanioum (cinnamon). The procedure was to mix the cold water and the gelatine, wait 5 min and then heat up the mixture to 45°C. 1050 g of the gelatine mixture was then poured into a blender and the mixture of CMC and AAB was added. The drops of cinnamon were added and the mixture was blended at the lowest speed for 3–5 s. It was then poured into pre-warmed molds and refrigerate for 36 h. Compressed air was used to remove the gelatine projectiles from the molds. Random samples across the length of two projectiles from every batch of gelatine made were used to verify the density and it was found consistently to be 1030 kg/m^3.

A2. Sabot development

A special sabot (Appendix B, Fig. B. 1) was designed and provides a unique efficient and inexpensive method to launch a soft body projectile. It was made of thermoplastic resin Acrylonitrile-Butadiene-Styrene (ABS) using additive manufacturing. This particular design allowed for the penetrator back end to be forward displaced in the sabot. If the projectile back end sits at the back of the sabot as it leaves the launch tube the compressive forces of the gelatine push the back end of the sabot apart causing the front ends to approach each other forming a pivot in front of the projectile. This provides an opportunity for the sabot to pass through the sabot trap and to strike the target. Displacing the projectile towards the front allows the compressive forces to start pushing the sabot apart while the back end is still in the launch tube and as it exits the aerodynamic forces continue to push the sabot sections apart and are subsequently stopped by the sabot trap while allowing the projectile to pass (Appendix B, Fig. B. 1(d)). The ribs around the circumference stiffen the sabot to avoid buckling in the launch tube.

A3. Shock pressure solution

When a shock is generated, the shock front in the material moves with a velocity U_s which is material dependent. The material behind a shock front moves with a particle velocity, u_p. The $U_s - u_p$ Hugoniot shown in Eq. (3) is normally derived experimentally [3,32] and have shown that for most materials it is a linear relationship (Appendix B, Fig. B. 4). If the $U_s - u_p$ data are available, as it is for most materials, the interface particle velocity and shock pressure could be obtained by using Eq. (3) in conjunction with the conservation equations for mass, momentum and energy given, respectively, by

$$\frac{\rho_1}{\rho_0} = \frac{v_0}{v_1} = \frac{U_s - u_0}{U_s - u_1} \tag{A.1}$$

$$P_1 - P_0 = \rho_0(P_1 u_1 - P_0 u_0)(U_s - u_0) \tag{A.2}$$

$$e_1 - e_0 = \frac{(P_1 u_1 - P_0 u_0)}{\rho_0(U_s - u_0)} + \frac{1}{2}(u_1^2 - u_0^2) \tag{A.3}$$

where the subscripts 0 and 1 are used to indicate conditions ahead and behind the shock front. Consider the impact problem described in Fig. 1 with the projectile moving from left to right for convenience. Combining Eq. (3) with the momentum Eq. (A.2), the $P - u$ Hugoniots [32,33], Eqs. (A.4) and (A.5), are obtained, respectively, for the right hand shock moving in the target, $u_p > u_0$, and the left hand shock moving in the projectile, $u_p < u_0$.

$$P_{ht} = \rho_{0t} c_{0t}(u_{1t} - u_{0t}) + \rho_{0t} s_t(u_{1t} - u_{0t})^2 \tag{A.4}$$

$$P_{hp} = \rho_{0p} c_{0p}(u_{0p} - u_{1p}) + \rho_{0p} s_p(u_{0p} - u_{1p})^2 \tag{A.5}$$

where the subscripts t and p relate, respectively, to the projectile and target materials. The pressure and the particle velocity across the interface of both materials must be consistent to satisfy the conservations equations. Thus, equating these two equations (shown graphically in Appendix B, Fig. B. 5) and eliminating the pressure terms results in a quadratic equation for the interface particle velocity, u_p given by

$$\begin{aligned}
(\rho_{0p} s_p - \rho_{0t} s_t)u_p^2 - \\
(\rho_{0p} c_{0p} + 2\rho_{0p} s_p u_{0p} + \rho_{0t} c_{0t} - 2\rho_{0t} s_t u_{0t})u_p + \\
(\rho_{0p} c_{0p} u_{0p} + \rho_{0p} s_p u_{0p}^2 + \rho_{0t} c_{0t} u_{0t} u_{0t} - \rho_{0t} s_t u_{0t}^2) = 0
\end{aligned} \tag{A.6}$$

Knowing the particle velocity provides three options to obtain the interface shock pressure either by Eq. (A.4) or (A.5) or by solving for the shock speed in Eq. (3) and then using the momentum equation(A.2) in which after the initial conditions are substituted, reduces to

$$P_h = \rho_0 U_s v_i \tag{A.7}$$

A4. Steady state solution

The steady state solution is obtained from the Alekseevskii–Tate [34–39] penetration equations

$$l\frac{dv_i}{dt} = \frac{-Y_p}{\rho_r} \tag{A.8}$$

$$\frac{1}{2}\rho_r(v_i - w)^2 + Y_p = \frac{1}{2}\rho_t w^2 + R_t \tag{A.9}$$

$$v_i = w - \frac{dl}{dt} \tag{A.10}$$

$$\frac{dP_n}{dt} = w \tag{A.11}$$

where v_i is the rod velocity, w is the penetration velocity, P_n is the penetration, l the penetrator length, t is the time after impact, R_t is the target strength and Y_p is the penetrator strength. ρ_p and ρ_t are, respectively, the penetrator and target densities. Given that the penetration velocity, $w = 0$, Eq. (A.10) could be solved explicitly for the projectile erosion

$$\frac{dl}{dt} = v_i \tag{A.12}$$

which implies essentially that the projectile erosion rate, $\frac{dl}{dt}$ is simply equal to impact velocity, v_i and the flow of the material at the target/projectile interface is considered to be steady state. Performing this simple integration and rearranging the equation making t the subject of the formula gives

$$t = l/v_i \tag{A.13}$$

which is the time the projectile spends interacting with the target before being completely eroded. In Eq. (A.9), setting the penetration velocity and taking the usual strength terms [36–39] as the pressures, an equation of the form

$$P_{interface} = \frac{1}{2}\rho_r v_i^2 + Y_p \tag{A.14}$$

For the case where the projectile strength is negligible, then $Y_p = 0$ and

$$P_{interface} = \frac{1}{2}\rho_p v_i^2 \tag{A.15}$$

is obtained where $P_{interface}$ is referred to as the stagnation pressure that is often found in literature [4–20].

Appendix B. Supplementary material

The supplementary material consists of five additional figures. Although the paper as it is could be read well without compromising the basic understanding of the work, the supplementary figures could be most helpful if the reader would like to use the test method, reproduce the results, follow in more detail the data reduction and analysis or the development of the interface shock pressure using the shock Hugoniot. Fig. B.1 shows various views of a unique sabot design that was used to launch the gelatine projectile and could be useful for reproducing or manufacturing the sabot. Fig. B. 2 shows the gelatine fragments recuperated and could shed some light on the failure of the material especially if one is considering of numerically simulating the impact problem using a hydrodynamic finite element code. Fig. B.3 shows the Force history in the frequency domain obtained using the Fourier transform in Matlab[25] and it does show the influence of natural vibrations of the target plates on the Force-time curves as
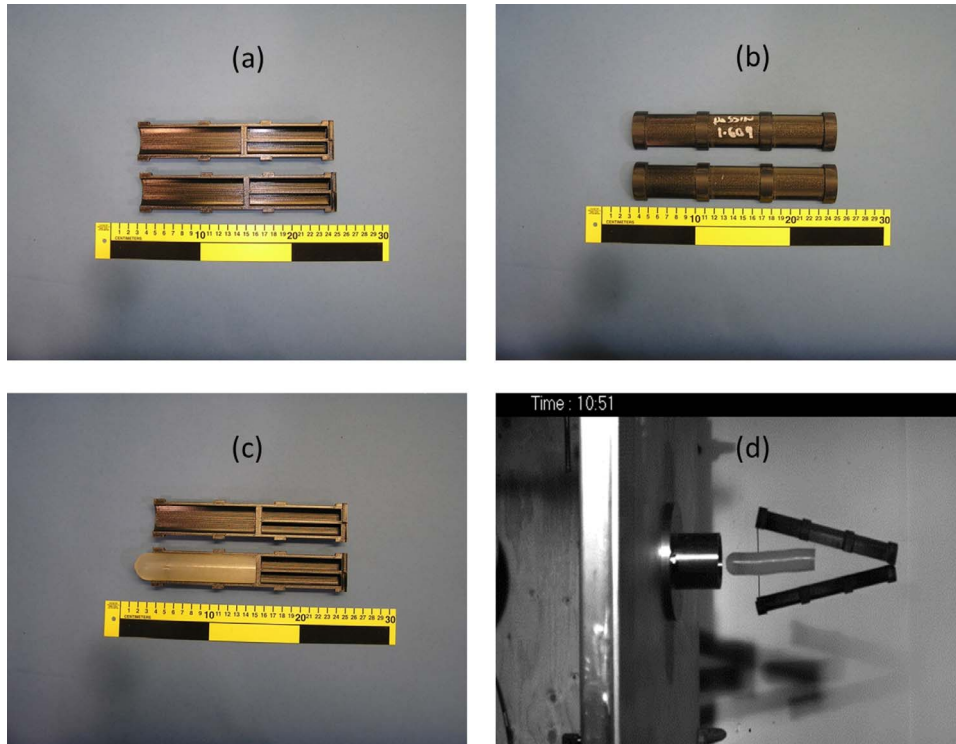

Fig. B.1. Various configuration views of the sabot: (a) the inside view. (b) a view of the outside (c) a view with the projectile placed in the sabot and (d) the separation of the projectile from the sabot just before passing through the sabot trap.
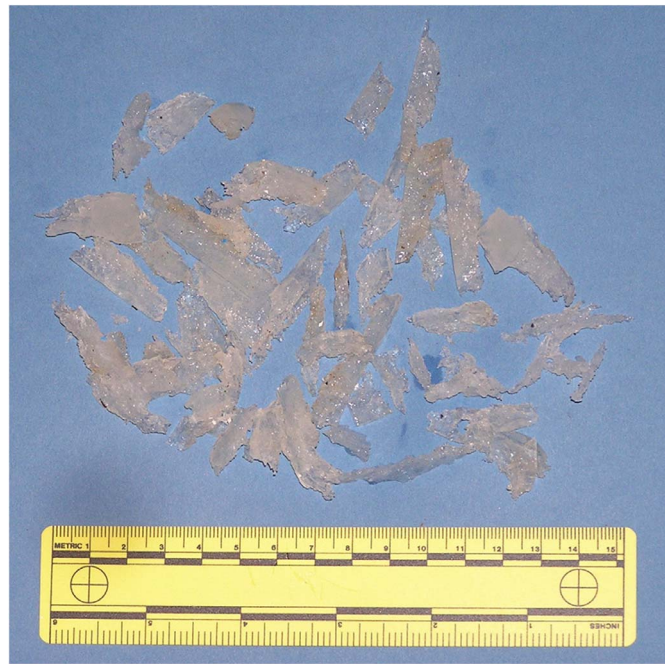
Fig. B.2. A collection of shredded gelatine fragments from the 119 m/s impact velocity test case.

a function of impact velocity. Fig. B. 4 shows a graphical representation of the $U_s - u_p$ shock Hugoniot, Eq. (3), that is used to obtain the shock speed for gelatine as a function of the particle velocity. Fig. B. 5 shows, for a particular impact velocity, a graphical solution for interface particle velocity by plotting the left hand and the right hand shock Hugoniots on the same axes. The intersection point of the two shock Hugoniots gives the interface particle velocity and shock pressure.
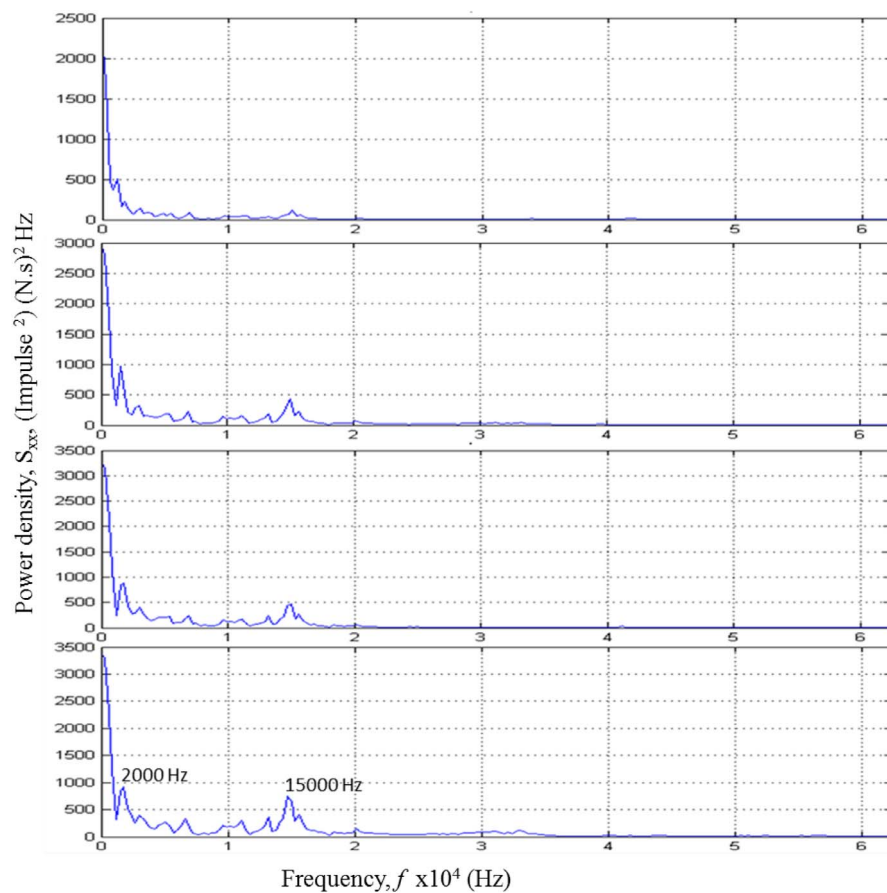

Fig. B.3. The force history in the frequency domain identifying frequencies present in the data.
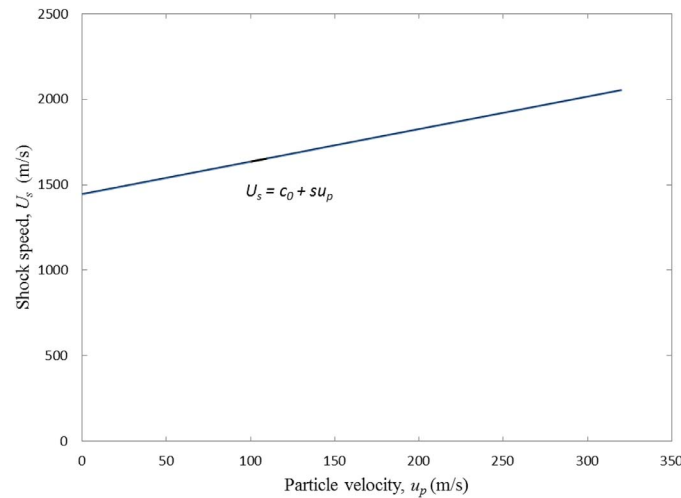
Fig. B.4. The $U_s - u_p$ shock Hugoniot for gelatine[3], $c_0 = 1445 \, \text{m/s}$, $s = 1.9$.
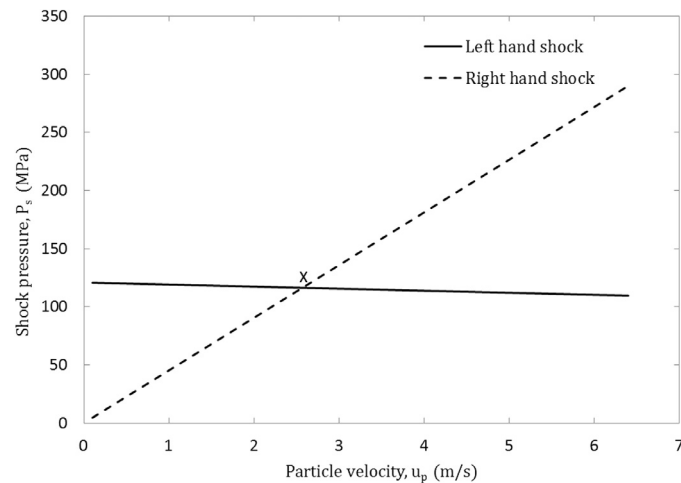

Fig. B.5. A graphical solution of simultaneously solving the left and right hand shock equations to obtain the interface shock pressure (point X) for a particular impact velocity, v_i.

Appendix C. Supplemetary material

Supplementary material associated with this article can be found, in the online version, at 10.1016/j.ijimpeng.2018.02.001.

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For a soft body projectile striking a target or the shock loading of a soft body material, the determination of the interface shock pressure, shock speed and applied steady state pressures is important but has been hindered by technical challenges even with the use of sophisticated embedded pressure sensors in the target surface. Difficulties interpreting the results render the accuracies sometimes questionable or impossible to reproduce. Here we propose a simple impact experiment using a force sensor and an analysis procedure to derive the interface pressure from the force/time history. The results are compared to those obtained from shock Hugoniot and penetration equations. We came upon the presence of a dynamic pressure that is significantly higher than the expected stagnation pressure. This method could be used to determine and characterise the shock and steady state pressures of a wider range of materials under impact and shock loading conditions.